



## **Challenges in the Design of Pump Turbines**

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Golden, CO, USA, November 2012



## **Challenges in the Design of Pump Turbines**

**High head, low specific speed**

- Head range: 1100–500 m
- Ternary or reversible machines
- 4 stage pump turbine (unregulated)  
to single stage reversible pump turbine

## Challenges in the Design of Pump Turbines

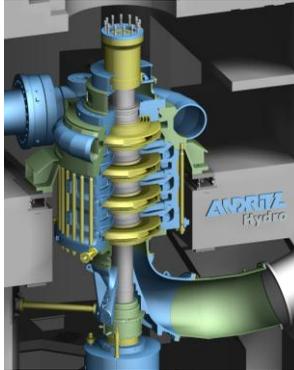
Application to projects: pump storage plant Tierfehd, Switzerland

### Pump turbine Tierfehd (Nestil)

1 reversible Pump turbine  
Customer: Axpo AG, Switzerland

4 stages, non-regulated  
Runner diameter  $D_1=2262$  mm  
Head range **H 953.4–1065.7 m**  
Max power  $P=141.2$  MW  
Speed  $n$  600 rpm  
Specific speed  $n_{sq}$  130

CFD and model tests well  
proven in site tests and experience  
Smooth operation behavior



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## Challenges in the Design of Pump Turbines

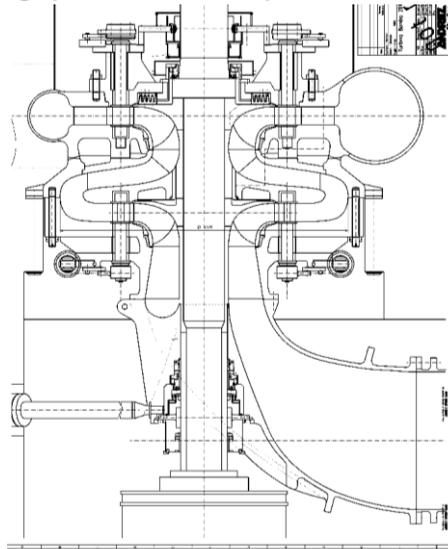
Application to projects: pump storage plant Moralets 2, Spain

### Moralets 2

2 Pump turbines  
Customer: Endesa, Spain

Two stages, both regulated  
Runner diameter  $D_1=2230$  mm  
Head range  $H$  700–804.8 m  
Max power  $P=202.6$  MW  
Speed  $n=750$  rpm  
Specific speed  $n_{sq}$  156

Engineering study  
Hydraulic development in progress  
Model test in 2013



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## Challenges in the Design of Pump Turbines

Application to projects: Pump storage plant Haeusling, Austria

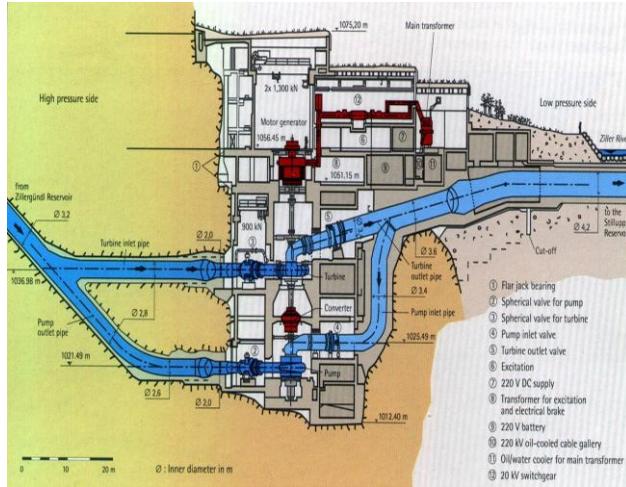
### Ternary plant Haeusling

2 Francis turbines

Customer:  
Verbund Hydro Power AG,  
Austria

Runner Diameter  $D_2=1800$  mm  
Head range H 568–734 m  
Max power  $P=175$  MW  
Speed  $n=600$  rpm  
Specific speed  $n_{sq}$  92

**World record head for  
Francis turbines  
In operation since 1986**



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## Challenges in the Design of Pump Turbines

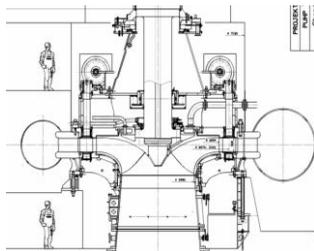
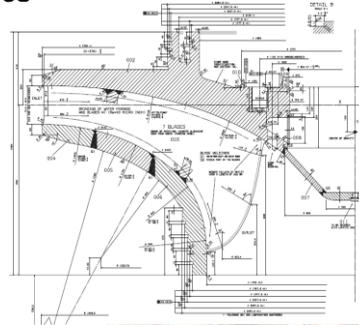
Application to projects: Bhira

### Bhira, India

1 Pump turbine, 1992

Customer: Tata Electric Companies, India

Runner Diameter  $D_1=3761$  mm  
Head range H 510–530m  
Max power  $P=165$  MW  
Speed  $n=500$  rpm  
Specific speed  $nsq$  90  
Runner band welded from three segments  
Successfully in operation



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## Challenges in the Design of Pump Turbines

### Application to projects: Hintermuhr

Customer: Salzburg AG, Austria

Runner outlet diameter  $D_1=1870$  mm

Head range H 455–517 m

Max power  $P=71.5$  MW

**Speed**  $n=1000$  rpm

Specific speed  $n_{sq}$  126

Scope of supply:

1 Pump Turbine

Motor Generator

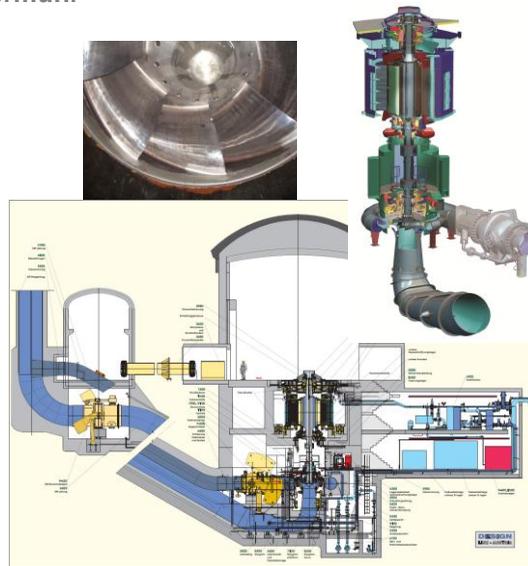
Governer

1 Butterfly valve

1 spherical valve

Existing cavern for two Pelton units

Guide vanes and labyrinth rings coated with tungsten carbide SXH70



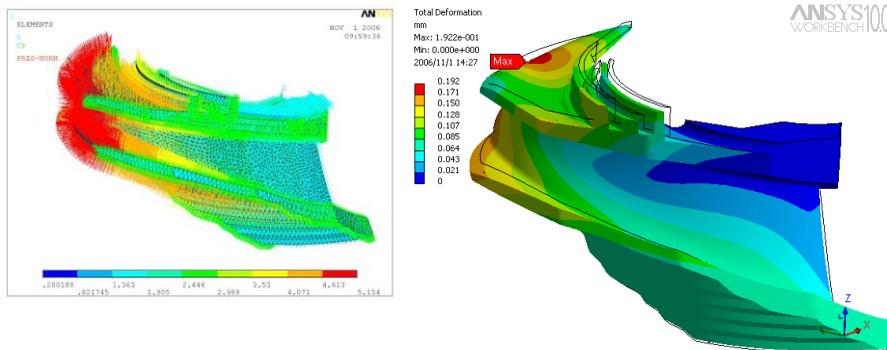
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## Challenges in the Design of Pump Turbines

### Structural integrity: Static analysis



Load on the runner: pressure field calculated with numerical flow simulation

Standard stress analysis for a pump turbine runner comprises 4 load cases

Load case 1: Pump operation at maximum head ( $Pu-H_{max}$ )

Load case 2: Pump operation at minimum head ( $Pu-H_{min}$ )

Load case 3: Turbine operation at maximum head ( $Tu-H_{max}$ )

Load case 4: Runaway, *Speed-no-Load*

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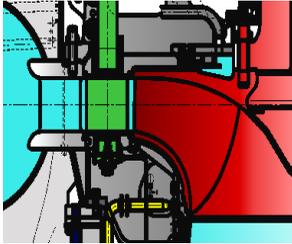
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## Challenges in the Design of Pump Turbines

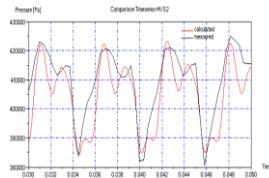
### Rotor/stator interaction in a pump turbine

#### Unsteady pressure fields from flow analysis

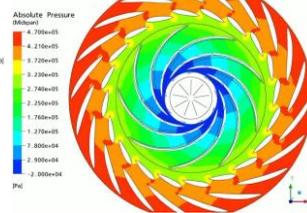
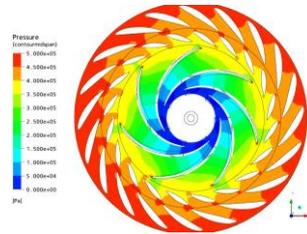


- Runner blades pass through wake of wicket gates
- Time-dependent pressure field
- Unsteady pressure distribution on runner blades
- Unfavorable blade number combinations can lead to higher pressure fluctuation

- 7 runner blades
- Pressure fluctuations in the 3. harmonic:  $21^*n$
- Rotating pressure mode with runner rotation: +Mode



- 9 runner blades
- Pressure fluctuations in the 2. harmonic:  $18^*n$
- Rotating pressure mode against runner rotation: -Mode



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## Challenges in the Design of Pump Turbines

### Rotor/stator interaction in a pump turbine

#### Dynamic analysis of the structure

Unsteady pressure fields from unsteady flow calculation for critical operating points

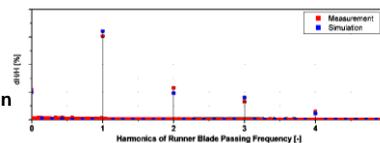
FEM analysis for specific excitation frequencies (harmonic response)

Pressure fields from CFD

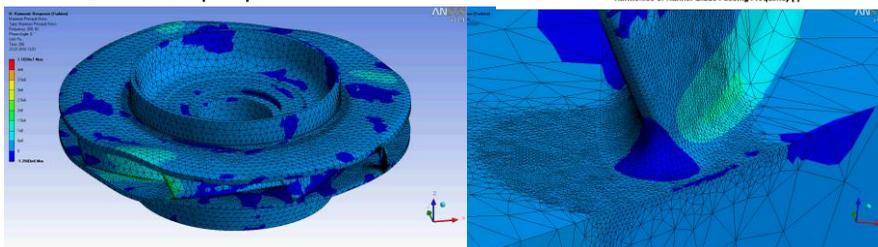
Time domain



Frequency domain



Distribution of principal stresses



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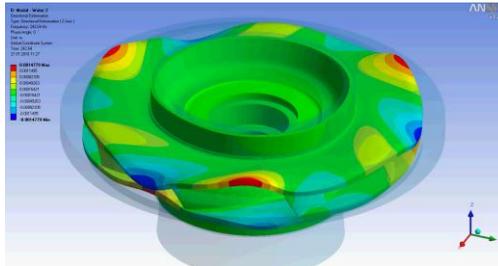
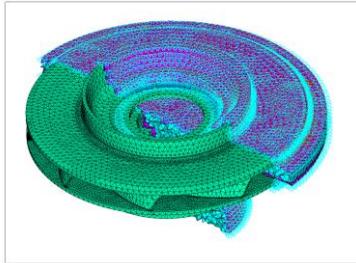
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## Challenges in the Design of Pump Turbines

### Dynamic analysis of the structure

#### Modal analysis



Nodal diameter 3: counter phase of hub and shroud

- Natural frequency of the runner considering casing, gaps and added mass of water
- Natural frequency of critical nodal diameter has to have safe distance from excitation frequency (blade passing frequency)
- Analysis of nodal diameters 1 to 10

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## Challenges in the Design of Pump Turbines

### Medium head, medium specific speed

- Head range: 200–500 m
- Reversible machines

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## Challenges in the Design of Pump Turbines

### Vianden M11

Customer: Societe Electrique de l'Our

Extension of existing pumped storage plant  
from 1100 MW to 1300 MW

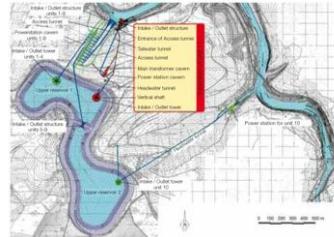
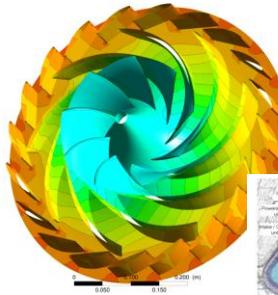
Runner Diameter  $D_1=4286$  mm

Head range  $H=269.4-294.6$  m

Max power  $P=200.4$  MW

Speed  $n=333.33$  rpm

Specific speed  $n_{sq}=156$



Scope of supply:

1 Pump Turbine

1 Motor/Generator, governor

1 Spherical valve

Draft tube gate

Single guide vane servo motors

Mechanical synchronization of guide vane

openings by synchronization ring

Hydraulically pre-stressed guide vane bearings

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## Challenges in the Design of Pump Turbines

Low head, high specific speed

- Head range:  $H < 200$  m
- Reversible machines
- Compared with low specific speed machines for same power:
  - Higher discharge
  - Bigger dimensions

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## Challenges in the Design of Pump Turbines

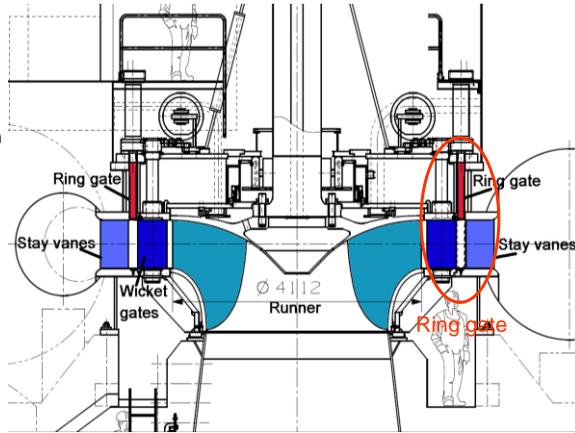
### Baixo Sabor Montante

Customer: EDP, Portugal

Runner Diameter  $D_1=4112$  mm  
 Head range  $H$  68.8–104.6 m  
 Max power  $P=76.9$  MW  
 Speed  $n=214.29$  rpm  
 Specific speed  $n_{sq}$  212

Scope of supply:  
 2 Pump Turbines  
 Motor/Generator  
 Governor  
 Ancillary equipment  
 Hydraulic steel structures

Wide head variation  
*Equipped with ring gate*



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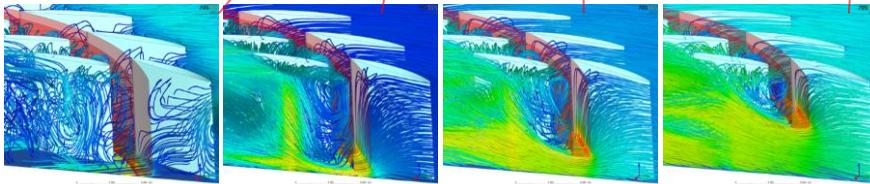
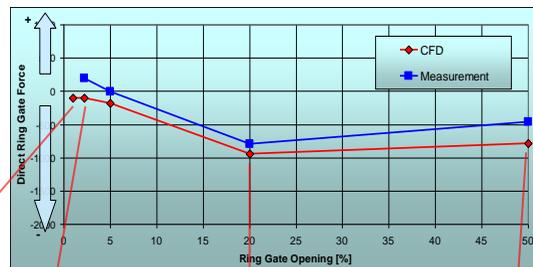
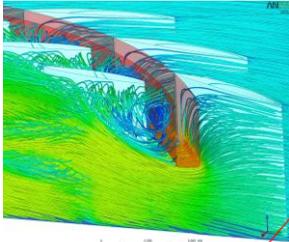
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## Challenges in the Design of Pump Turbines

### Ring gate

- Axial force on gate
- Comparison flow simulation - measurement



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## Challenges in the Design of Pump Turbines

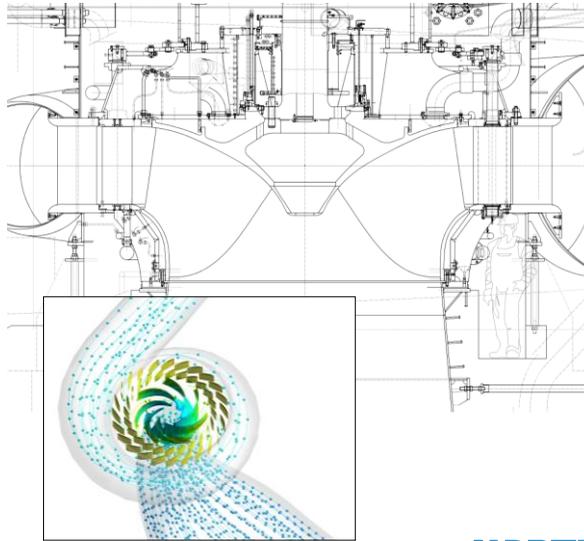
### Baixo Sabor Jusante

Customer: EDP, Portugal

Runner Diameter  $D_1=3948$  mm  
Head range H 26.2–35.2 m  
Max power  $P=17.8$  MW  
Speed  $n=150$  rpm  
Specific speed  $n_{sq}$  264

Scope of supply:  
2 Pump turbines  
Motor/generator  
Governor  
Ancillary equipment  
Hydraulic steel structures

Wide head variation



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## Challenges in the Design of Pump Turbines



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- Pumped storage plants can vary significantly in size, head and discharge
- In most cases, the electro-mechanical equipment (turbines and generators) is custom made specific to the site
- Specific technical challenges depending on size, head and discharge
- Technical challenges relate to hydraulic performance and mechanical integrity
- Design processes have to consider the operating regime (e.g. number starts and stops)

Thank you for your attention

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